



Problem (1)

(A) Choose the correct answer:

- a- Helical gears are:
 - 1- has low working noise than spur gears
 - 2- same noise as spur gears
 - 3- has high working noise than spur gears
- b- Gear module equals:
 - 1- N/d where N is number of teeth and d is the pitch circle diameter
 - 2- $d \times N$
 - 3- d/N
- c- Rigidity of shafts is related to :
 - 1- Resulted stresses
 - 2- Resulted lateral defection
 - 3- Resulted twisting angle
- d- Rolling bearings can sustain:
 - 1- Bending and reaction forces
 - 2- Torsion and reaction forces
 - 3- Reaction forces only
- e- rolling bearings has friction :
 - 1- more than sliding bearings
 - 2- less than sliding bearings
 - 3- equal to sliding bearings
- f- For the same power transmission, spur gears will have design factor of safety:
 - 1- Equal to helical gear
 - 2- More than helical gear of same size
 - 3- Less than helical gear
- g- Flat belt has standard size and length:
 - 1- Correct
 - 2- False
- h- The bending stress educed in gears teeth must be compared to:
 - 1- Teeth surface hardness
 - 2- Teeth strength
 - 3- Material creep
- i- The effect of buckling must be taken into consideration when the shaft is subjected to:
 - 1- Axial compression load
 - 2- Axial tension load

j- In the front wheel drive cars, the front axle is considered as:

- 1- Shaft
- 2- Not shaft

(B) If SKF bearing with dynamic capacity ($C = 20 \text{ kN}$ and $C_o = 16 \text{ kN}$) is used for a pump shaft rotates with **1000 RPM** and under axial load = **0.5 kN** and Radial load = **10 kN**, determine its expected life in working hours.

Problem (2)

The following figure shows a water pump operated by an electric motor through an open type **V-belt** drive. The motor pulley diameter = **100 mm**, pump pulley diameter = **300 mm**, Coefficient of friction for belt with pulleys = **0.32**, Center distance between the two pulleys = **850 mm**; Rotational speed of the motor = **1500 RPM**; Power transmission = **20 kW**; allowable stress for the belt material (S_{all}) = **2 MPa**; thickness of the belt = **4 mm**.

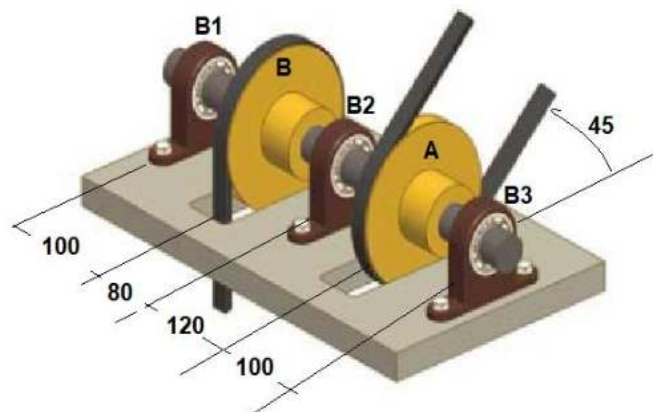
- a- Find the belt length
- b- Find the belt tensioning forces
- c- If the belt permissible tension force is not to exceed **800 N**, how many belts will you need?



Problem (3)

A countershaft carrying two flat-belt pulleys is shown in the following figure. Pulley **A** receives a torque of **300 Nm** from a motor through a belt as shown in figure below. The power is transmitted through the shaft and delivered to the belt on pulley **B**. The belt tensioning forces on pulley **A** are **1800** and **270 N** and the belt tensioning forces on pulley **B** are **3000** and **552 N**.

- (a) Draw the bending-moment diagram for the shaft.
- (b) Find the safety factor of the shaft (allowable shear strength = **40 MPa**)



Helpful Data

For Belts and Pulleys

$$\theta_1 = 180 - 2\sin^{-1}\left(\frac{D-d}{2C}\right)$$

$$\theta_2 = 180 + 2\sin^{-1}\left(\frac{D-d}{2C}\right)$$

$$L = \sqrt{4C^2 - (D-d)^2} + \frac{1}{2}(D\theta_2 + d\theta_1)$$

$$(F_1 - F_2)\frac{d}{2} = T$$

$$\frac{F_1}{F_2} = e^{\mu\theta}$$

Bearing Selection

$$F_e = XF_r + YF_a$$

$$L = \left[\frac{C}{F_e}\right]^3$$

F_a/C_0	e	$F_a/(VF_r) \leq e$		$F_a/(VF_r) > e$	
		X_1	Y_1	X_2	Y_2
0.014*	0.19	1.00	0	0.56	2.30
0.021	0.21	1.00	0	0.56	2.15
0.028	0.22	1.00	0	0.56	1.99
0.042	0.24	1.00	0	0.56	1.85
0.056	0.26	1.00	0	0.56	1.71

ASME Eq. for shaft design

$$d_o = \left[\frac{16}{\pi S_{s\text{allow.}} (1 - \lambda^4)} \sqrt{\left(k_b M_b + \frac{\alpha F d_o (1 + \lambda^2)}{8} \right)^2 + (K_t M_t)^2} \right]^{1/3}$$

$$\theta = \frac{TL}{GJ} \quad (\text{rad.})$$