



**Problem (1) Choose the correct answer:**

- a- Spur gears are:
  - 1- has low working noise than helical gears
  - 2- same noise as helical gears
  - 3- has high working noise than helical gears
- b- Gear module equals:
  - 1-  $N/d$  where N is number of teeth and d is the pitch circle diameter
  - 2-  $d \times N$
  - 3-  $d/N$
- c- Rigidity of shafts is related to :
  - 1- Resulted lateral defection
  - 2- Resulted stresses
  - 3- Resulted twisting angle
- d- Rolling bearings can sustain:
  - 1- Torsion and reaction forces
  - 2- Bending and reaction forces
  - 3- Reaction forces only
- e- Sliding bearings has friction :
  - 1- more than rolling bearings
  - 2- less than rolling bearings
- f- For the same power transmission, the helical gears have design factor of safety:
  - 1- Equal to spur gears of same size
  - 2- More than spur gear of same size
  - 3- Less than spur gear of same size
- g- V-belt has standard size and length:
  - 1- Correct
  - 2- False
- h- The bending stress educed in gears teeth must be compared to:
  - 1- Teeth surface hardness
  - 2- Teeth strength
  - 3- Material creep
- i- The shaft rigidly depends on:
  - 1- Axial load
  - 2- Bending load
  - 3- Torque load
- j- In the 4-wheel drive cars, the real axle is considered as:

- 1- Shaft
- 2- Not shaft

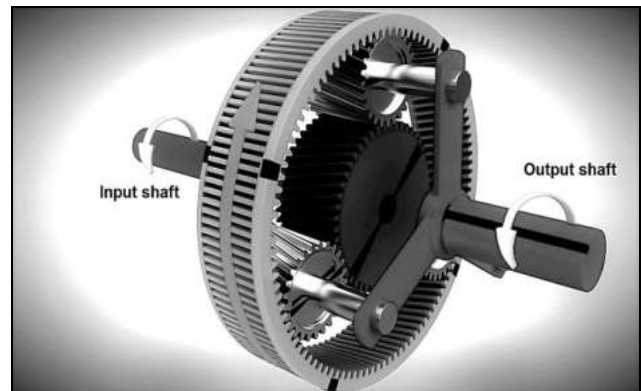
**(B)** A simply supported shaft, diameter **40 mm**, on bearing supports carries a radial load of **20 kN** at its center. The axial load on the bearings is **1.1 kN**. The shaft speed is **1500 RPM**. Find the bearing working hours if the bearing dynamic and static capacities are **28 kN** and **20 kN**, respectively.

### Problem (2)

The following figure shows a planetary gear box used to transmit power of **10 kW**. The speed of the input shaft is **1500 RPM (CW)**. The number of teeth of gears are as follows:

Sun gear has **20** teeth, Each planet gear has **30** teeth, all gears have **4 mm** module. Find the following:-

- a- The number of teeth on the ring gear.
- b- The speed (and direction) of the output shaft, *the ring gear is kept fixed*.
- c- Torque applied on sun gear
- d- Bending stress resulting in teeth of the sun helical gear (all the gears have **45 mm** width). (take  $K_o=K_m=1.3$ )

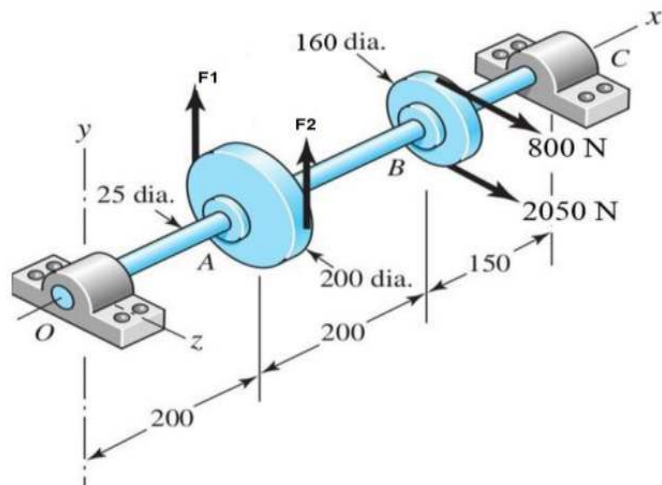


### Problem (3)

The following figure shows a shaft transmits power from the shown pulley at **A** to the pulley at **B** (of diameter = **160 mm**). The shaft rotates at **1500 RPM**. Find the following:

- a- Transmitted power between the two pulleys.
- b- Belt forces at Pulley **A** ( $F_1$  and  $F_2$ ), take belt friction coefficient = **0.3**.
- c- Shaft diameter at critical section, if its material is **St.52** ( $S_{ut} = 520 \text{ MPa}$ ) and safety factor = **3** based on yielding strength.

Dimensions are in mm



## Helpful Data

### Bearing Selection

$$F_e = XVF_r + YF_a$$

$$L = \left[ \frac{C}{F_e} \right]^3$$

$F_a/C_0$	$e$	$F_a/(VF_r) \leq e$		$F_a/(VF_r) > e$	
		$X_1$	$Y_1$	$X_2$	$Y_2$
0.014*	0.19	1.00	0	0.56	2.30
0.021	0.21	1.00	0	0.56	2.15
0.028	0.22	1.00	0	0.56	1.99
0.042	0.24	1.00	0	0.56	1.85
0.056	0.26	1.00	0	0.56	1.71

### Gear Design Data

The bending stress equation for helical gear teeth is given as

$$\sigma = \frac{F_t}{bmJ} K_v K_o (0.93 K_m)$$

$$K_v = \left[ \frac{78 + (200V)^{0.5}}{78} \right]^{0.5}$$

### ASME Eq. for shaft design

$$d_o = \left[ \frac{16}{\pi S_{s \text{ allow.}} (1 - \lambda^4)} \sqrt{\left( k_b M_b + \frac{\alpha F d_o (1 + \lambda^2)}{8} \right)^2 + (K_t M_t)^2} \right]^{1/3}$$

$$\theta = \frac{TL}{GJ} \quad (\text{rad.})$$