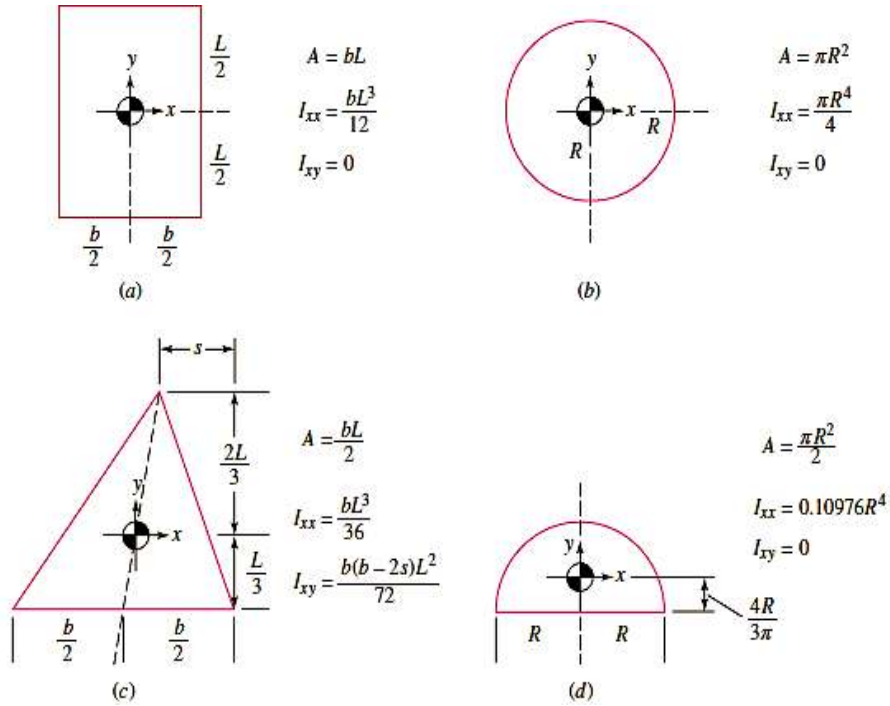


Thursday 27/7/1442 (11/3/2021)

Time Allowed: 1:30 hours

Name: _____ **Number:** _____ **Section:** _____



Density of water, $\rho = 1000 \text{ kg/ m}^3$

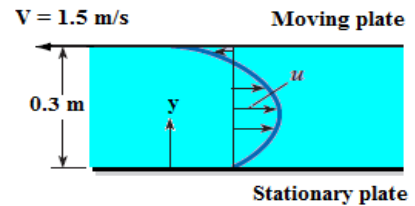
Problem #1

A Laminar flow of viscous fluid ($\mu = 1.44 \text{ kg/m} \cdot \text{s}$) occurs between two horizontal parallel plates. The upper plate moves to the left at velocity $V = 1.5 \text{ m/s}$. The expression for local velocity $u(y)$ is given as

$$u(y) = 60(0.3y - y^2) - 5y$$

Where y is the vertical coordinate from the bottom surface. **Calculate:**

- The shear stress at the stationary plate and the moving plate.
- The position of zero shear stress and the velocity at that position.



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(a) Shear stress (τ) = $\mu \frac{du}{dy}$.

→ at the stationary plate, $y=0$ and $\tau = \tau_s$

$$\tau_s = \mu \left. \frac{du}{dy} \right|_{y=0}$$

$$\frac{du}{dy} = 60[0.3 - 2y] - 5$$

$$\left. \frac{du}{dy} \right|_{y=0} = 60[0.3 - 0] - 5 = 18 - 5 = 13/s$$

$$\tau_s = \mu \left. \frac{du}{dy} \right|_{y=0} = 1.44 \times 13 \text{ N/m}^2 = 18.72 \text{ N/m}^2 = 18.72 \text{ Pa}$$

$\tau_s = 18.72 \text{ Pa}$ Shear stress at stationary plate.

→ at moving plate, $y=0.3 \text{ m}$ and $\tau = \tau_m$.

$$\left. \frac{du}{dy} \right|_{y=0.3} = 60[0.3 - 2(0.3)] - 5 = -18 - 5 = -23/s$$

$$\tau_m = \mu \left. \frac{du}{dy} \right|_{y=0.3} = 1.44(-23) \text{ Pa} = -33 \text{ Pa}$$

$\tau_m = -33 \text{ Pa}$ Shear stress at the moving plate.

(b) Shear stress equation:-

$$\tau = \mu \frac{du}{dy} = \mu [60(0.3 - 2y) - 5]$$

To find position of zero stress:- put $\tau = 0$.

$$\mu [60(0.3 - 2y) - 5] = 0.$$

$$60(0.3 - 2y) - 5 = 0$$

$$60(0.3 - 2y) = 5.$$

$$0.3 - 2y = \frac{5}{60} = 0.0833.$$

$$0.3 - 0.0833 = 2y.$$

$$\frac{0.2167}{2} = y$$

$$\boxed{0.1083 \text{ m} = y} \rightarrow \text{Position of Zero Shear Stress.}$$

Velocity at that position is:-

$$U_0 = 60[0.3(0.1083) - (0.1083)^2] - 5(0.1083) \quad \text{at } y = 0.1083 \text{ m.}$$

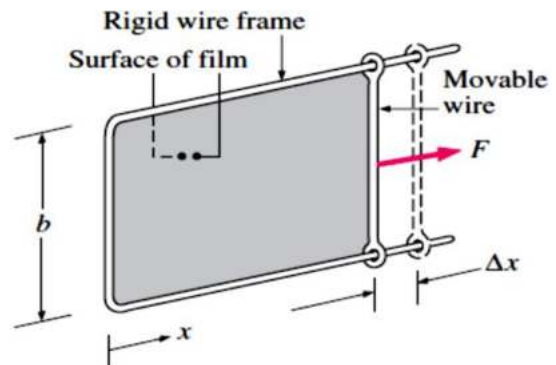
$$U_0 = 1.2456 - 0.5415 \text{ m/s}$$

$$\boxed{U_0 = 0.7041 \text{ m/s}} \text{ velocity at position } (y = 0.1083 \text{ m}) \text{ of zero shear stress.}$$

Problem #2

The surface tension of a liquid is to be measured using a liquid film suspended on a U-shaped wire frame with an 8-cm-long movable side (b).

If the force needed to move the wire is **0.012 N**, determine the surface tension of this liquid in air .



Step 1 of 1 ^

Surface tension formula $\sigma_s = \frac{F}{2b}$

Where F: Required force

b: movable length

Substitute: $b = 8 \text{ cm} = 0.08 \text{ m}$, $F = 0.024 \text{ N}$

$$\sigma_s = \frac{0.024 \text{ N}}{2 \times 0.08 \text{ m}}$$

$$= 0.15 \text{ N/m}$$

Thus surface tension 0.15 N/m

Handwritten solution:

$$F = F_p$$

$$F = \gamma \times p$$

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$$0.012 = \gamma \times 2 \times 0.08$$

$$\gamma = \frac{0.012}{2 \times 0.08} = 0.075 \text{ N/m}$$

OR

$$\gamma = \frac{F}{p} = \frac{F}{2L}$$

$$= \frac{0.012}{2 \times 0.08} = 0.075 \text{ N/m}$$

Problem #3

A steady, incompressible, two-dimensional velocity field is given by the following components in the xy-plane:

$$u = 1.1 + 2.8x + 0.65y, \quad v = 0.98 - 2.1x - 2.8y$$

- Calculate**
- The acceleration field components (a_x and a_y),
 - The acceleration at the point $(-2, 3)$.

Solⁿ:-

$$\vec{V} = (2.8x + 0.65y + 1.1)\hat{i} + (0.98 - 2.1x - 2.8y)\hat{j}$$

$$\Rightarrow u = 2.8x + 0.65y + 1.1$$

$$\& v = 0.98 - 2.1x - 2.8y$$

(a) At stagnation point

$$u = 0 \quad \& \quad v = 0$$

$$2.8x + 0.65y + 1.1 = 0 \quad \text{--- (i)}$$

$$0.98 - 2.1x - 2.8y = 0 \quad \text{--- (ii)}$$

by solving equation (i) & (ii)

$$x = -0.2654$$

$$y = -0.549$$

So stagnation point = $(x, y) = (-0.2654, -0.549)$

(b)

$$v_x = u = u(-1.5, 2.0) = 2.8x + 0.65y + 1.1$$

$$= 2.8(-1.5) + 0.65(2) + 1.1$$

$$\boxed{v_x = -1.8 \text{ m/sec.}}$$

$$v_y = v(-1.5, 2.0) = 0.98 - 2.1x - 2.8y$$

$$= 0.98 - 2.1(-1.5) - 2.8(2.0)$$

$$\boxed{v_y = 9.73 \text{ m/sec.}}$$

$$a_x = u \cdot \frac{\partial u}{\partial x} + v \cdot \frac{\partial u}{\partial y} = (2.8x + 0.65y + 1.1)(2.8) + (0.98 - 2.1x - 2.8y)(0.65)$$

$$a_x(-1.5, 2.0) = -1.8 \times 2.8 + 9.73 \times 0.65$$

$$\boxed{a_x = 1.2845}$$

$$a_y = u \cdot \frac{\partial v}{\partial x} + v \cdot \frac{\partial v}{\partial y} = (2.8x + 0.65y + 1.1)(-2.1) + (0.98 - 2.1x - 2.8y)(-2.8)$$

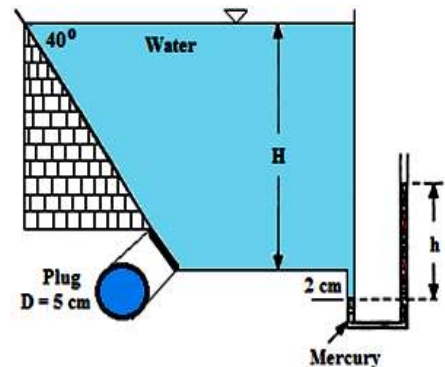
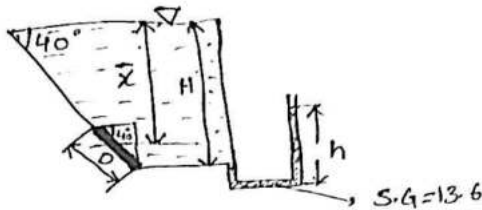
$$a_y(-1.5, 2.0) = (-1.8)(-2.1) + (9.73)(-2.8)$$

$$\boxed{a_y = 31.024}$$

Problem # 4

The tank in the figure has a 5-cm diameter plug. If the hydrostatic force on the plug is 30 N, Calculate:

- The water height in the tank H and
- The reading h of Mercury (SG = 13.6) the manometer

STEP-1:a) GIVEN:

$$\text{DIAMETER OF PLUG (D)} = \frac{5}{100} = 0.05 \text{ m}$$

$$\text{HYDROSTATIC FORCE (F)} = 30 \text{ N}$$

FOR PLUG:FROM ΔABC

$$\sin 40^\circ = \frac{BC}{D/2}$$

$$BC = \frac{D}{2} \sin 40^\circ$$

$$\therefore \bar{x} = H - BC$$

$$= H - \frac{D}{2} \sin 40^\circ$$

 \therefore HYDROSTATIC FORCE (F)

$$F = \rho_{\text{water}} \times A \times \bar{x}$$

$$= \rho_{\text{water}} \times g \times \frac{\pi}{4} (d^2) \times \left(H - \frac{D}{2} \sin 40^\circ\right)$$

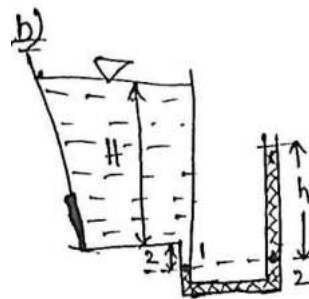
$$= 1000 \times 9.81 \times \frac{\pi}{4} (0.05)^2 \times \left(H - \frac{0.05}{2} \sin 40^\circ\right)$$

$$F = 19.262 \times (H - 0.324)$$

$$\text{BUT } F = 30 \text{ N}$$

$$\therefore 19.262 (H - 0.324) = 30$$

$$\therefore H = 1.57355 \text{ m}$$



EQUATING PRESSURES AT ① & ②

$$P_1 = P_2$$

$$\rho_{\text{water}} \times g \left(H + \frac{2}{100}\right) = \rho_{\text{mercury}} \times g \times h$$

$$\left(H + \frac{2}{100}\right) = \frac{\rho_{\text{mercury}}}{\rho_{\text{water}}} \times h$$

$$H + \frac{2}{100} = 13.6 \times h$$

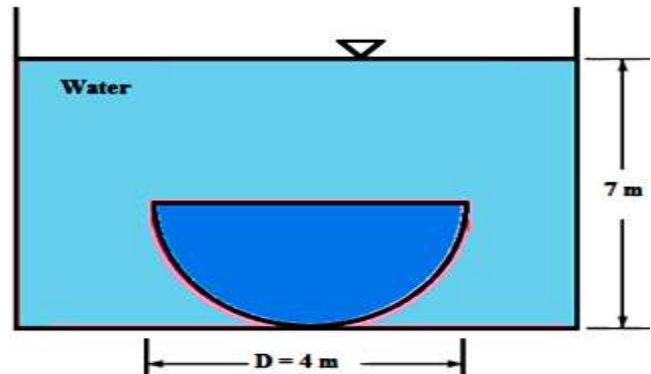
$$1.57355 + 0.02 = 13.6 \times h$$

$$\therefore h = 0.117173 \text{ m}$$

$$h = 11.7173 \text{ cm}$$

Problem # 5

A hemispherical object (SG = 7.8) is submerged in a water tank as shown in the figure. Determine the net hydrostatic force on the hemispherical object.



for water $W = 9800 \text{ N/m}^3$

depth $[h(y)] = -y$

$$x^2 + y^2 = R^2$$

$$x = \sqrt{4 - y^2}$$

$$\therefore L = 2x$$

$$L(y) = 2\sqrt{4 - y^2}$$

$$F = W \int_c^d L(y) h(y) dy$$

$$= 9800 \int_{-2}^0 -y (2\sqrt{4 - y^2}) dy$$

$$= 9800 \left[\frac{2(4 - y^2)^{3/2}}{3} \right]_{-2}^0$$

$$= \frac{9800 \times 2}{3} \left[(4 - 0)^{3/2} - (4 - 4)^{3/2} \right]$$

$$= 52266.67 \text{ N}$$

$$= \underline{52.27 \text{ kN}} \text{ Answer}$$