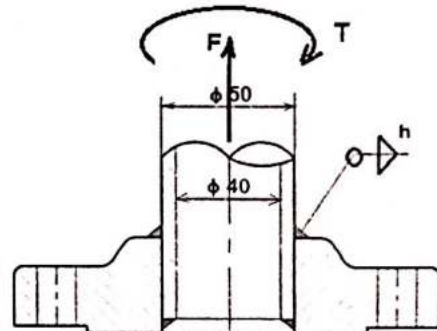




Student Name: I.D. #:

Problem (1) [8 Marks]

For the welded joints shown in figure below, find the weld size (h) under the following applied loads, (if its allowable shear strength = **100 MPa**).
Axial force, $F = 50$ kN and Torsional load, $T = 2$ kN.m



$$\tau_{d.s.} = \frac{F}{0.707h \Sigma l}$$

$$= \frac{50,000}{2 \times 0.707h (2\pi \times 25)} = \frac{225.22}{h} \text{ N/mm}^2$$

$$\tau_T = \frac{T r}{J} = \frac{T r}{2 \times 0.707h J_w}$$

$$= \frac{2 \times 10^6 \times 25}{2 \times 0.707h (2\pi \times 25^3)}$$

$$= \frac{360.36}{h} \text{ N/mm}^2$$

$$\tau_{Max.} = \tau_{d.s.} + \tau_T = \frac{585.6}{h} \leq 100 \Rightarrow h \geq 5.85 \text{ mm}$$

Torsional Properties of Fillet Welds*

Weld	Throat Area	Location of G	Unit Second Moment of Area
5.	$A = 1.414h(b + d)$	$\bar{x} = b/2$ $\bar{y} = d/2$	$J_G = \frac{b^3 + d^3}{6}$
6.	$A = 1.414 \pi r$		$J_G = 2\pi r^3$

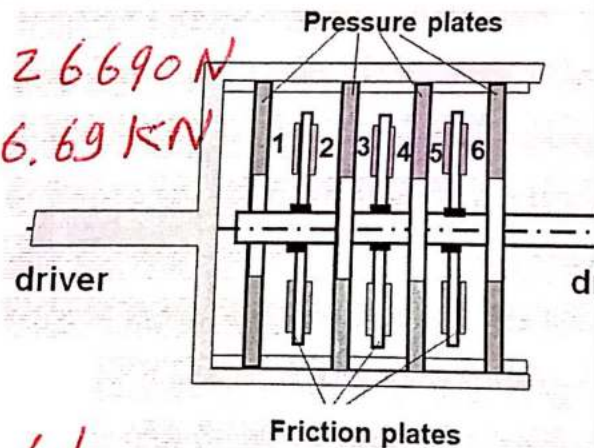
Problem (2) [6 Marks]

An engine has its rated output power of **100 HP**. The clutch used is of multi-plate type as shown in figure below. The maximum contact pressure of the clutch lining is **850 kPa**. The external diameter and internal diameters of each friction plate are **250** and **150 mm**, respectively. Determine the maximum torque which can be transmitted by this clutch and at what speed. Coefficient of friction of clutch lining with flywheel = **0.35**.

$$F_a = 0.850 \pi (125^2 - 75^2) = 26690 \text{ N} \\ = 26.69 \text{ kN}$$

$$T_b = 6 \times 26690 \times 100 \times 0.35 \\ = 56049 \text{ N.m}$$

$$\omega = \frac{\text{power}}{T} = 13.38 \text{ rad./s}$$



$$F_a = P_c \pi (R_o^2 - R_i^2) \\ T_b = Z F_a R_m$$

Problem (3) [6 Marks]

The band brake shown in figure below is actuated by force **W**. The band has a face width of **40 mm** and a mean coefficient of friction of **0.35**. If the maximum contact pressure between the band and drum is not to exceed **0.49 MPa**, find the following:

- a- The band forces (**F₁** and **F₂**).
- b- The braking torque.
- c- Actuating force **W**.

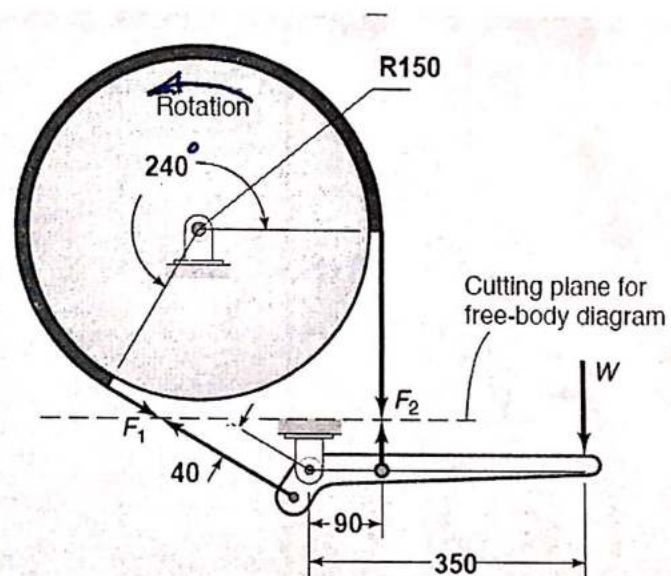
$$\frac{F_2}{F_1} = e^{\mu \theta} = e^{0.35 \times \frac{240\pi}{180}} = 4.286$$

$$p = \frac{F_2}{bR} = \frac{F_2}{40 \times 150}$$

$$F_2 = 2940 \text{ N}$$

$$F_1 = 685.95 \text{ N}$$

$$T_b = (F_2 - F_1)R = (2940 - 685.95) \times 0.150 = 338 \text{ N}$$



$$W \times 350 = F_2 \times 90 - F_1 \times 40$$

$$W = 677.6 \text{ N}$$