

Q1

Given Data

$$T = 200^\circ\text{C}$$

$$N = 500,000 \text{ cycles}$$

$$R = 90\%$$

$$f = 0.9$$

$$r = 0.06''$$

$$d = 0.6''$$

$$\Delta = 0.72''$$

Material Properties

From table A-20:

$$\left. \begin{array}{l} S_{ut} = 64 \text{ Kpsi} \\ S_{yt} = 54 \text{ Kpsi} \end{array} \right\} \text{ for AISI } 1018 \text{ CD Steel}$$

Solution

$$(a) \quad \frac{r}{d} = 0.1, \quad \frac{\Delta}{d} = \frac{0.72}{0.6} = 1.2, \quad q = 0.73$$

↳ From Fig. 6-20

$$K_t = 1.87 \text{ from table A-15-14}$$

$$\Rightarrow K_f = 1 + q(K_t - 1) = 1 + 0.73(1.87 - 1) \quad \text{1 mark}$$

$$\therefore K_f = 1.635$$

Marin's factors → critical point is at groove

$$\Rightarrow K_a = a S_{ut}^b = 2.7 \times (64)^{-0.265} \text{ from table 6-2}$$

$$K_a = 0.897 \text{ (for CD steel)}$$

$$d_e = 0.37d = 0.37 \times 0.6 = 0.222 \rightarrow \text{For non-rotating completely reverse bending}$$

$$\Rightarrow K_b = 0.879 d_e^{-0.107} = 0.879 (0.222)^{-0.107}$$

$$\therefore K_b = 1.033 \text{ for } 0.11 < d_e < 2 \text{ in.}$$

$$\Rightarrow K_c = 1.0 \text{ for pure bending}$$

2 marks

$$\Rightarrow K_d = \frac{S_T}{S_{RT}} = 1.020 \text{ from table 6-4.}$$

$$\Rightarrow K_e = 0.897 \text{ from table 6-5}$$

$$\Rightarrow K_f = 1.0, \text{ no any miscellaneous factor considered}$$

$$\Rightarrow S_e = K_a K_b K_c K_d K_e K_f S_e' \rightarrow \text{Marin's equation}$$

$$S_e = 0.897 \times 1.033 \times 1.0 \times 1.02 \times 0.897 \times 1.0 \times 0.5 \times 64$$

$$S_e = 27.13 \text{ Kpsi}$$

2 marks

Hence, Constant a and b are given as:

$$a = \frac{(f S_{ut})^2}{S_e} = \frac{(0.9 \times 64)^2}{27.13} = 122.29 \text{ Kpsi}$$

2 marks

$$b = -\frac{1}{3} \log \left(\frac{f S_{ut}}{S_e} \right) = -\frac{1}{3} \log \left(\frac{0.9 \times 64}{27.13} \right) = -0.109$$

$$\Rightarrow \sigma_{rev} = S_f = a N^b \rightarrow n=1$$

$$\sigma_{rev} = 122.29 (500,000)^{-0.109}$$

$$\sigma_{rev} = 29.26 \text{ Kpsi}$$

2 marks

But $\sigma_{rev} = K_f \frac{32M}{\pi d^3}, M = \pm F_y \cdot l = \pm 2.5 F_y$

$$\Rightarrow 29.26 = 1.635 \times \frac{32 \times 2.5 F_y}{\pi (0.6)^3}$$

2 marks

$$\therefore F_y = 151.8 \text{ lbf} \rightarrow \text{Maximum Completely reversed force}$$

⑥ $F_y = \pm 100 \text{ lbf}$, $M = \pm 100 \times 2.5 = 250 \text{ lbf} \cdot \text{in}$

2 marks

$$\sigma_{rev} = K_f \times \frac{32M}{\pi d^3} = 1.635 \times \frac{32 \times 250}{\pi (0.6)^3}$$

$$\sigma_{rev} = 19.275 \text{ Kpsi}$$

From (a), $S_e = 27.13 \text{ Kpsi}$

$$\Rightarrow n = \frac{S_e}{\sigma_{rev}} = \frac{27.13}{19.28} = 1.41$$

2 marks

↓
Factor of Safety
for infinite life
if $F_y = \pm 100 \text{ lbf}$

Q2 Given Data

$$W = 2.5''$$

$$t = 3/8''$$

$$r = 0.25''$$

$$n = 2$$

$$d = W - 2r = 2.0''$$

Material properties

$$S_{ut} = 68 \text{ kpsi}$$

$$S_{yt} = 57 \text{ kpsi}$$



- From table A-20

- For AISI 1020 CD Steel

Solution

$$\frac{W}{d} = \frac{2.5}{1.25} = 1.25$$

2 marks

→ $K_t = 2.25$ from Figure A-15-3

$$\frac{r}{d} = \frac{0.25}{1.25} = 0.125$$

→ critical point is at groove.

The notch sensitivity, q , is to be determined either from Fig. 6-20 or Eqs. (6-34) and (6-35).

However, the $r = 0.25''$ is not covered by the graph. Therefore, we will use Eq. (6-35a):

$$\sqrt{a} = 0.246 - 3.08 \times 10^{-3} S_{ut} + 1.51 \times 10^{-5} S_{ut}^2 - 2.67 \times 10^{-8} S_{ut}^3$$

↳ For axial load

$$\Rightarrow a = 9.601 \times 10^{-3}$$

$$\Rightarrow q = \frac{1}{1 + \frac{\sqrt{a}}{\sqrt{r}}} = 0.836$$

2 marks

$$\Rightarrow K_f = 1 + q(K_t - 1) = 1 + 0.836(2.25 - 1) = 2.045$$

$$\Rightarrow \sigma_{rev} = K_f \cdot \frac{F_a}{A} \quad \text{and} \quad n = \frac{S_e}{\sigma_{rev}} \rightarrow \text{For infinite life}$$

⇒ Marin's factors

$$\Rightarrow K_a = 2.70(68)^{-0.265} = 0.88$$

$$\Rightarrow K_b = 1 \text{ (axial loading)} \quad \text{2 marks}$$

$$\Rightarrow K_c = 0.85 \text{ (axial loading)}$$

$$K_d = 1, \text{ assuming room temperature}$$

$$K_e = K_f = 1, \text{ not specified}$$

$$\Rightarrow S_e = K_a \cdot K_b \cdot K_c \cdot K_d \cdot K_e \cdot K_f \cdot S_e'^{15} \Rightarrow S_e' = 0.5 S_u$$

$$\Rightarrow S_e = 0.88 \times 0.85 \times 0.5 \times 68 = 25.4 \text{ Kpsi}$$

2 marks

$$\Rightarrow \bar{\sigma}_{rev} = \frac{S_e}{n} = \frac{25.4}{2} = 12.7 \text{ Kpsi}$$

$$\Rightarrow 12.7 = 2.045 \times \frac{F_a}{3/8 \times (2.0)}$$

$$\therefore F_a = 4.6567 \text{ kips}$$

2 marks